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"Flexible transporting apparatus for presses"

Description

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The invention relates to a press line or multi-stage press for large components, having a transporting apparatus for transporting workpieces, according to the preamble of Claim 1.

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Prior art

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In a press, press line or multi-stage press for large components, transfer apparatuses are provided for transporting workpieces into the processing stages. In recent systems according to EP 0 672 480 B1 or EP 0 693 334 A1, the transporting operation between individual processing stations takes place individually by individual transporting apparatuses, which allow, in particular, a high flexibility of the capacity for movement of the workpiece transportation between individual processing stages. By means of such a drive, which is fully independent of the central drive of the press, it is possible to optimize the transportation of the workpiece in a number of degrees of freedom, in particular in relatively large press installations. For this purpose, you are referred to EP 0 672 480 or EP 0 693 334. By way of example, carrying rails, on which carriages with dedicated drive travel, are provided over the entire press length. For accommodating the workpieces, use is made of crossmembers which are provided with retaining means and are each fastened on 2 opposite carriages. In the most straightforward embodiment, 2 transporting movements are provided for

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transferring the workpieces, to be precise a vertical movement and a horizontal movement. The vertical movement serves for removing the workpiece from the bottom die part or depositing the workpiece in the same, while the horizontal movement provides the actual transporting step. This transporting step can take place from one press into the following press or, in the case of a multi-stage press for large components, from one forming station into the next.

However, it is usually the case that the workpieces and/or dies are not of such straightforward configuration as to allow transportation in biaxial operation. By way of example, in the case of passenger-vehicle doors, the latter, in the first forming stage, are drawn from a common blank in order then, following a cutting operation, to run, each as separate workpieces, through the processing stages together. In order to avoid more expensive and complicated dies, it is necessary for the workpiece to be brought into an optimum processing position during the transfer operation. This change in position is usually carried out by way of intermediate set-down locations or orienting stations.

Such an intermediate set-down location, both for single and for double components, is disclosed by EP 0 383 168 B1 or DE 196 51 934 A1. Of particular note are the 5 degrees of freedom which can be used for changing the position of workpieces of complex configuration. It is thus possible, if required, for the position of the workpiece to be manipulated in 5 axes.

Essential disadvantages of this functionally satisfactory intermediate set-down location are as follows:

- the press installation or multi-stage press for large components requires a long overall length since the intermediate set-down locations are arranged between the processing stages and the

appropriate amount of space thus has to be provided.

- The number of workpiece-specific changeover parts is high.
- 5 - The parked position of the crossmembers during the forming operation is restricted.
- The cycle speed and functional reliability of the press may be adversely affected by the relatively large number of transporting steps.

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This resulted in considerations to dispense with the intermediate set-down location and to integrate the necessary degrees of freedom in the transporting systems. It is thus proposed, [lacuna] DE
15 44 08 449 A1, to configure the transporting system such that the crossmember can be brought into a sloping position in the vertical direction. It is additionally possible to pivot an axis in the direction transverse to the transporting direction.

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Some of the possible movements of the intermediate set-down location have thus been integrated in the transporting system, but the full functionality of this intermediate set-down location has not.

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Object and advantage of the invention

Taking the prior art as the departure point, the object of the invention is to propose a
30 transporting system for forming machines which has the highest possible number of degrees of freedom or movement axes.

This object is achieved, taking as the departure point a transporting system according to the
35 preamble of Claim 1, by the characterizing features of Claim 1. Advantageous and expedient developments of the transporting system are specified in the subclaims.

The invention is based on the idea of configuring a separately driven transfer for each die

stage such that workpieces can undergo an optimum change in position adapted to the forming process in each case.

By way of example, the change in position may include the following movement axes:

- horizontal displacement in and counter to the transporting direction
- sloping position in the transporting direction
- 10 - displacement in the direction transverse to the transporting direction
- pivoting in and counter to the transporting direction
- pivoting in the direction transverse to the transporting direction
- 15 - vertical change in height

By a different combination of the movements, the change in position is made possible during introduction of the workpieces into the die and removal of the workpieces from the die.

Provision is made here to ensure the functionality both for individual large-surface-area workpieces and for 2 workpieces, that is to say so-called double components.

In the case of the design, taking as departure point the known individually driven, crossmember-bearing transporting systems, such as carriages, slides, pivoting arm, telescopic arm, etc., the number of movement axes is increased by additional drives and movement-transmissions. By using spherical mountings, such as ball and socket joints or universal joints, a sloping position of the crossmember is also made possible.

Further details and advantages of the invention can be gathered from the following description of an exemplary embodiment.

The higher-outlay solution of transporting double components has been selected for the exemplary

embodiment. If, however, the task is to transport just one large-surface-area workpiece rather than a double component, the crossmember is replaced by the sucker crossmember. This function is achieved by the attachment of die-specific transporting and retaining means to the crossmember.

In the figures:

Figure 1 shows part of a multi-stage press for large components,

Figures 2a, 2b show a view of a transporting unit in the direction transverse to the transporting direction, Figures 3a, 3b show a detail from Figure 2, and Figures 4a, 4b show a plan view of the transporting unit.

Description of an exemplary embodiment

Processing stations or forming stages 8, 9 of a multi-stage press for large components 1 are illustrated in Figure 1. Arrow 30 shows the transporting direction of the workpieces. The transporting apparatus 2 is arranged on the press upright 3 and also mirror-invertedly on the opposite upright. The transporting apparatus 2 is driven by pivot drive 6, which is in operative connection with pivoting arm 4. The crossmember, which is provided for workpiece-transporting purposes, is designated 5 and is mounted on the pivoting arm 4. This figure shows, in particular, the following degrees of freedom

- vertical movement
- horizontal movement
- crossmember 5 pivot [sic] in and counter to the transporting direction

Via the pivoting drive 6, in operative connection with a lifting drive 7, by virtue of a combination of movements, a transporting curve or a transporting step comprising vertical and horizontal movements is executed. The transporting step serves for

transferring the workpiece from, for example, forming stage 8 to forming stage 9. If a change in position of the workpieces, on account of different removal and feeding positions, and thus better introduction and delivery is necessary, the crossmember 5 can be pivoted about the axis of rotation 12. A drive 10 causes the crossmember 5 to pivot via a toothed-belt drive 11. Different positions of the crossmember 5 can clearly be seen in Figure 1.

10 The illustration in Figures 2a + b shows crossmember 5 in a horizontal position and in a vertically sloping position. The figures show the mutually opposite arrangement of the transporting apparatuses 2.1 and 2.2 with fastening on the left-hand and right-hand uprights 3.1 and 3.2. Movably arranged workpiece-specific sucker crossmembers 13 for transporting double components are provided on the crossmember 5 by way of example in Figures 2a + b. It is also possible, without any restrictions, to use just one centrally arranged sucker crossmember 13, as is necessary, for example, for transporting a large, not yet divided blank or a large workpiece. In this case, the suckers are connected directly, as changeover parts, to crossmember 5. A transverse-displacement movement may be provided.

 The following degrees of freedom are illustrated in Figures 2a + b:

- 30 - pivoting the sucker crossmember 13 in the direction transverse to the transporting direction in the case of double components
- horizontal and sloping position in the vertical direction of the crossmember 5.

35 The pivoting of the sucker crossmember 13 is described in more detail in Figures 3a + b.

 The vertically sloping position of the crossmember 5 is achieved by different movement sequences of transporting apparatus 2.1 and 2.2. For

the compensation in length which is required by the sloping position according to Figure 2b, a spline shaft 14 is provided. The universal joint 15 allows the angled position of the crossmember 5. Instead of a universal joint 15, an axis of rotation is also initially sufficient for this sloping position.

Figures 3a + b show design details for pivoting the sucker crossmember 13. The following is also illustrated as a further degree of freedom:

- transverse displacement of the sucker crossmember 13

Figures 3a + b show the end of the pivoting arm 4 of the transporting apparatus 2 with the mount for the crossmember 5. The toothed-belt drive 11 is integrated in the transporting apparatus 2 in order to pivot the crossmember 5 about the axis of rotation 12. The spline shaft 14, on the one hand, transmits the rotational movement and, in addition, allows the compensation in length for the sloping position of the crossmember 5. The spline shaft 14 is fastened to the universal joint 15. The pivotable bearing block 17 bears drives 18, 19, which drive spindle/nut system 20 and 21 via shafts and angular gear mechanisms. Rods 23 arranged on both sides are in operative connection with spindle/nut system 20 and are connected to circle segment 24 and pivot the latter at the point of rotation 25. The maximum size of the pivoting angle is $W1$ and $W2$. The circle segment 24 is guided and supported by segment guides or guide rollers 26, which are fastened on horizontal slide 27. The workpiece-retaining sucker crossmember 13 is connected to the circle segment 24. Guides 28 serve for guiding the horizontal slide 27. Said horizontal slide 27 can be displaced by the distance $M1$ and $M2$ in relation to its central position. Horizontal slide 27 is driven, via rod 29 and spindle/nut system 21, by drive 19. This apparatus described may be fitted on the crossmember 5

on its own or as one of two. The combination of movements is possible by simultaneous actuation of the drives 18, 19. The rotational-speed regulation may result in the same or different rotational speeds, as a result of which optimum conditions for handling the workpieces are achieved. This high flexibility may also be advantageous during die changeover, where, if appropriate, it is possible to dispense with the exchange of the component-specific sucker crossmember 13 and to execute just a horizontal movement. If, however, an exchange of the sucker crossmembers 13 is necessary, then all the movement elements on crossmember 5 remain.

The crossmember 5 can be disengaged at the separating location 22, as may be necessary, for example, during a conversion from a double component to a large-surface-area single component. Advantageously, in the arrangement proposed, there is no need to exchange the drives 18, 19, and these remain in the press 1.

A combination of pivoting and horizontal displacement of the sucker crossmember 13 is not absolutely necessary in every case. Alternatively, the attachment may be such that only one movement is possible in each case, i.e. the slide 27 or the circle segment 24 may then be dispensed with.

Figures 4a + b show a plan view of crossmember 5 in a horizontal position and a horizontally sloping position in the component-transporting direction in accordance with arrow 30. The double-sided arrangement of the actuating rods 23, 29 and, in extension thereof, the pivoting and transverse-displacement apparatus are illustrated. Two sucker crossmembers 13 are likewise attached.

Pairs of the actuating rods 23, 29 are fitted in each case on the spindle/nut systems 20, 21, which are provided as a single unit. Figure 4b shows the following further degree of freedom:

- horizontally sloping positioning about the vertical axis in or counter to the transporting direction

5 If it is only this sloping position which is required, the function can be performed with an axis of rotation and the compensation in length by spline shaft 14. If, however, the vertically sloping position described in Figure 2 is likewise envisaged, then the
10 use of a universal joint 15 is necessary. By virtue of this design solution, any desired combination of vertically and horizontally sloping positions is also possible, and thus an

- sloping positioning in space
- 15 is provided as the further degree of freedom.

 In its maximum inventive configuration, the transporting system proposed may thus carry out workpiece manipulation in the following degrees of freedom.

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- Vertical movement upward and downward
 - horizontal movement in and counter to the transporting direction
 - pivoting of the crossmember and sucker crossmember
25 in and counter to the transporting direction
 - vertically sloping positioning of the crossmember and sucker crossmember
 - pivoting of the sucker crossmember in the direction transverse to the transporting direction
 - 30 - transverse displacement of the sucker crossmember
 - horizontally sloping positioning of the crossmember and sucker crossmember in and counter to the transporting direction
 - sloping positioning of the crossmember and sucker
35 crossmember in space

 The invention is not restricted to the exemplary embodiment which has been described and illustrated. It also covers all expert configurations

within the scope of the applicable Claim 1. Thus, a universal joint is only to be understood by way of example as a movable mounting, and it is possible to use all spherical joints which satisfy the requirements of the inventive idea.

As has been explained, it is possible, during the transportation of single components, to dispense with a separate pivotable sucker crossmember 13 and to use crossmember 5 directly as sucker crossmember.

List of designations:

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| | 1 | Multi-stage press for large components |
| | 2 | Transporting apparatus |
| 5 | 3 | Press upright |
| | 4 | Pivoting arm |
| | 5 | Crossmember |
| | 6 | Pivoting drive |
| | 7 | Lifting drive |
| 10 | 8 | Forming stage |
| | 9 | Forming stage |
| | 10 | Drive |
| | 11 | Toothed-belt drive |
| | 12 | Axis of rotation |
| 15 | 13 | Sucker crossmember |
| | 14 | Spline shaft |
| | 15 | Universal joint |
| | 17 | Bearing block |
| | 18 | Drive |
| 20 | 19 | Drive |
| | 20 | Spindle and nut |
| | 21 | Spindle and nut |
| | 22 | Separating location |
| | 23 | Rod |
| 25 | 24 | Circle segment |
| | 25 | Point of rotation |
| | 26 | Guide |
| | 27 | Horizontal slide |
| | 28 | Guide |
| 30 | 29 | Rod |
| | 30 | Component-transporting apparatus |